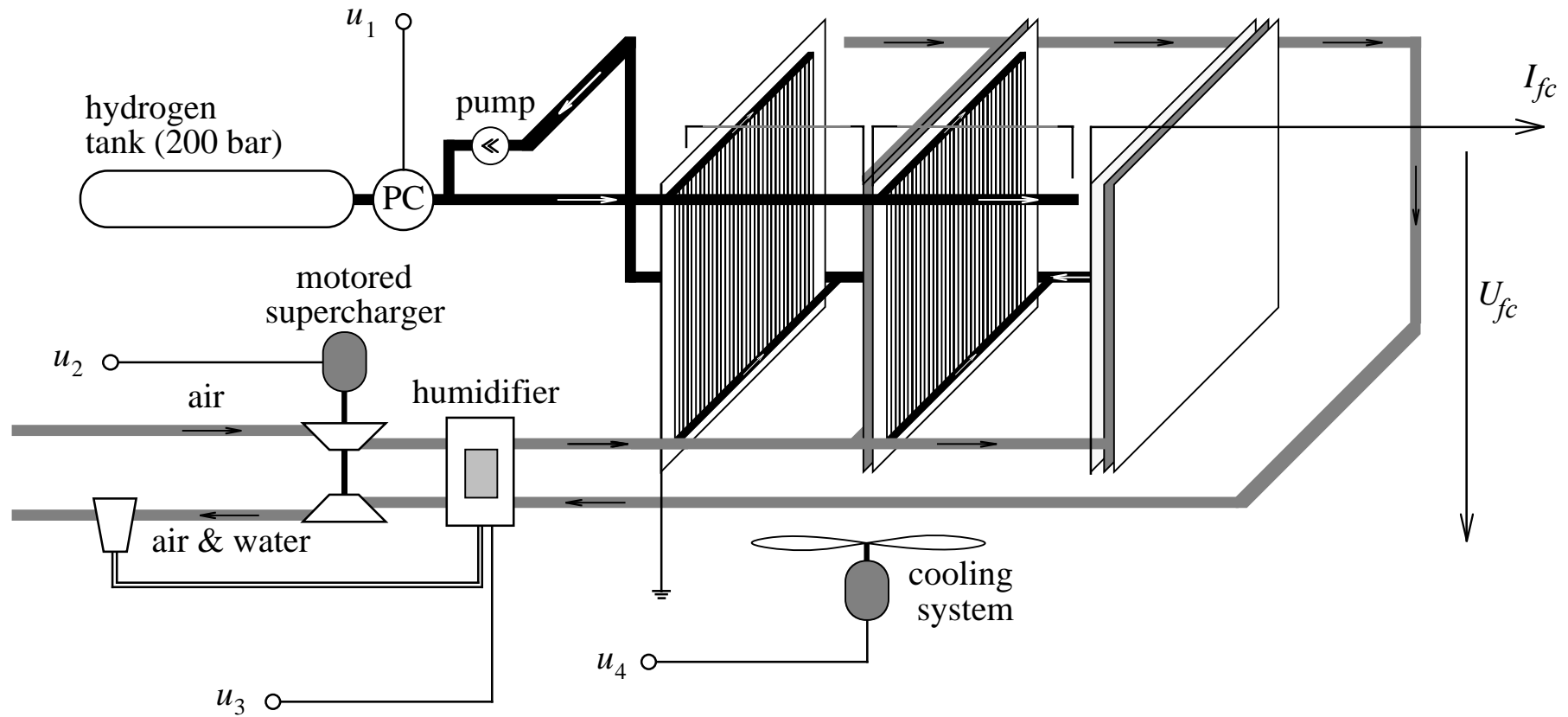
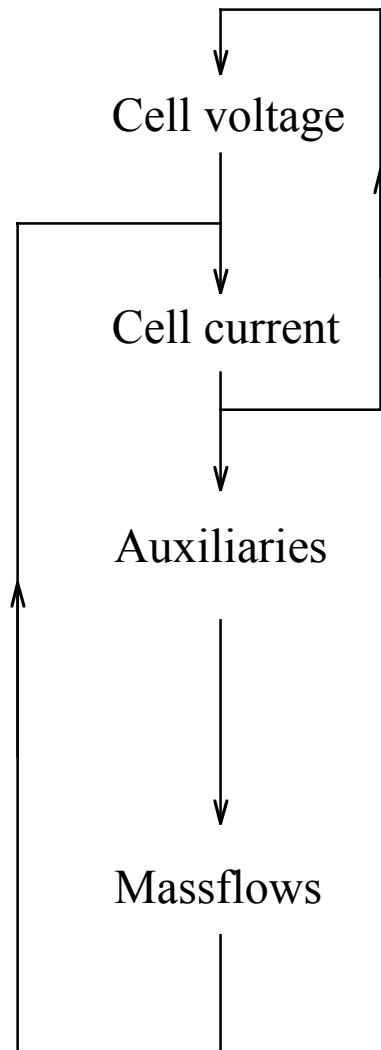


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$$U_{fc} = f(I_{fc}) \cdot N$$

Solve implicit loops using affine approximation for the cell voltage and noting that the auxiliary power is typically linear in the cell current

$$I_{fc} = \frac{P_{el} + P_{aux}}{U_{fc} \cdot N}$$

P_{el} = input
(desired net power)

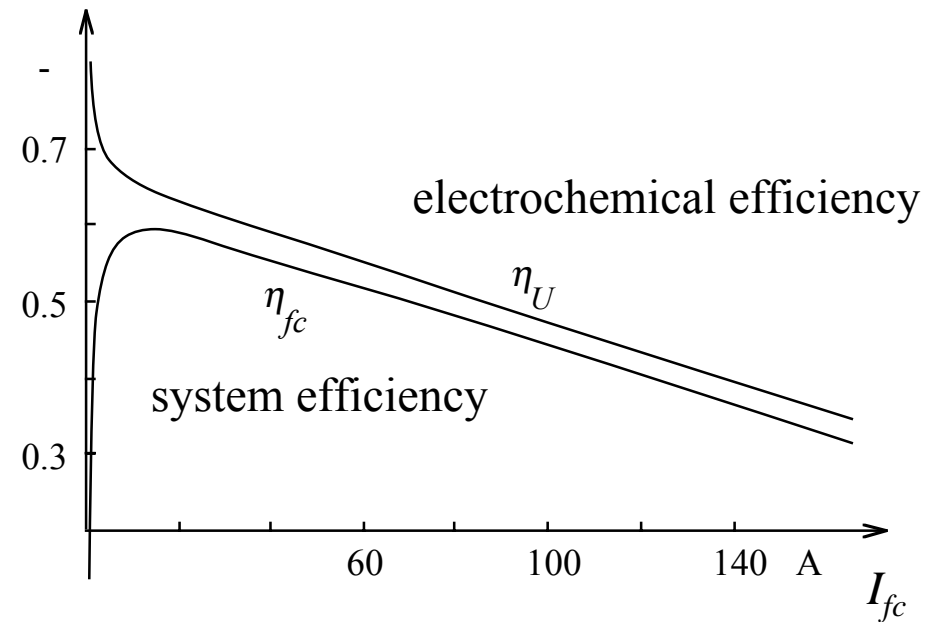
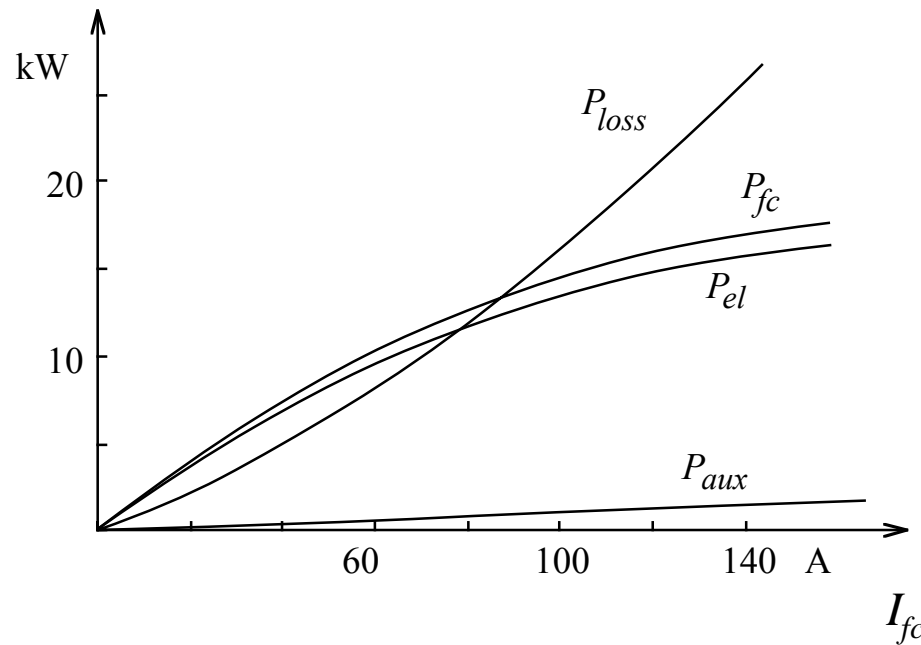
$$P_{aux} = g(\dot{m}_{air}, \dot{m}_{H2})$$

e.g.
$$P_{aux,air} = \dot{m}_{air} \cdot c_{p,air} T_1 \left(\pi^{(\kappa_{air}-1)/\kappa_{air}} - 1 \right) \frac{1}{\eta_C \eta_{EM}}$$

$$\dot{m}_{air} = \lambda_{air} \cdot N \cdot \frac{M_{air} / 0.21}{4 \cdot F} \cdot I_{fc} \quad \lambda_{air} \geq 1 \text{ (typically 1.5 ... 2)}$$

$$\dot{m}_{H2} = \lambda_{H2} \cdot N \cdot \frac{M_{H2}}{2 \cdot F} \cdot I_{fc} \quad \lambda_{H2} \geq 1 \text{ (typically 1.1 ... 1.2)}$$

Result:
$$I_{fc} = \frac{N(U_0 - k_{aux}) - \sqrt{N^2(U_0 - k_{aux})^2 - 4NR_{fc}P_{el}}}{2NR_{fc}}$$



- Remarks:
- Cooling not trivial (several 10 kW at rated loads), water-cooling requires de-ionized water
 - FC efficiency at low loads very poor due to auxiliaries

Dynamic effects (estimates of the time constants or delays)

$O(10^{-9}s)$ electrochemistry (dissociation and recombination kinetics)

$O(10^{-9}s)$ RC-element (electrode/membrane system)

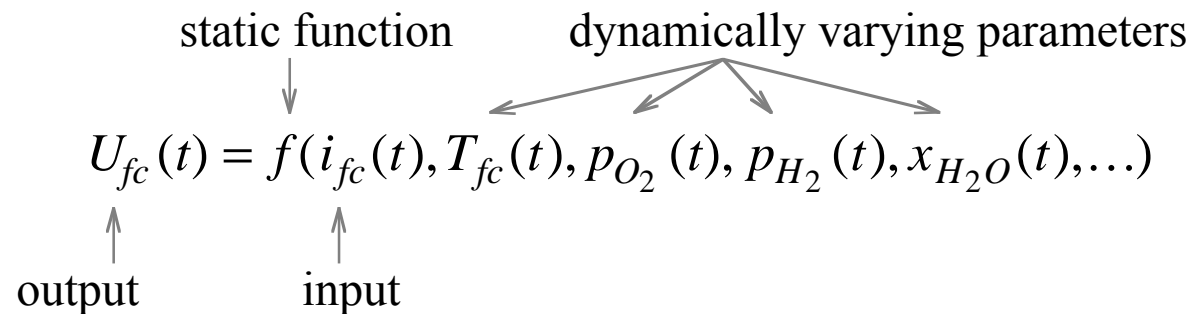
$O(10^{-1}s)$ hydrogen and air manifolds (mass storages)

$O(10^0s)$ membrane water content

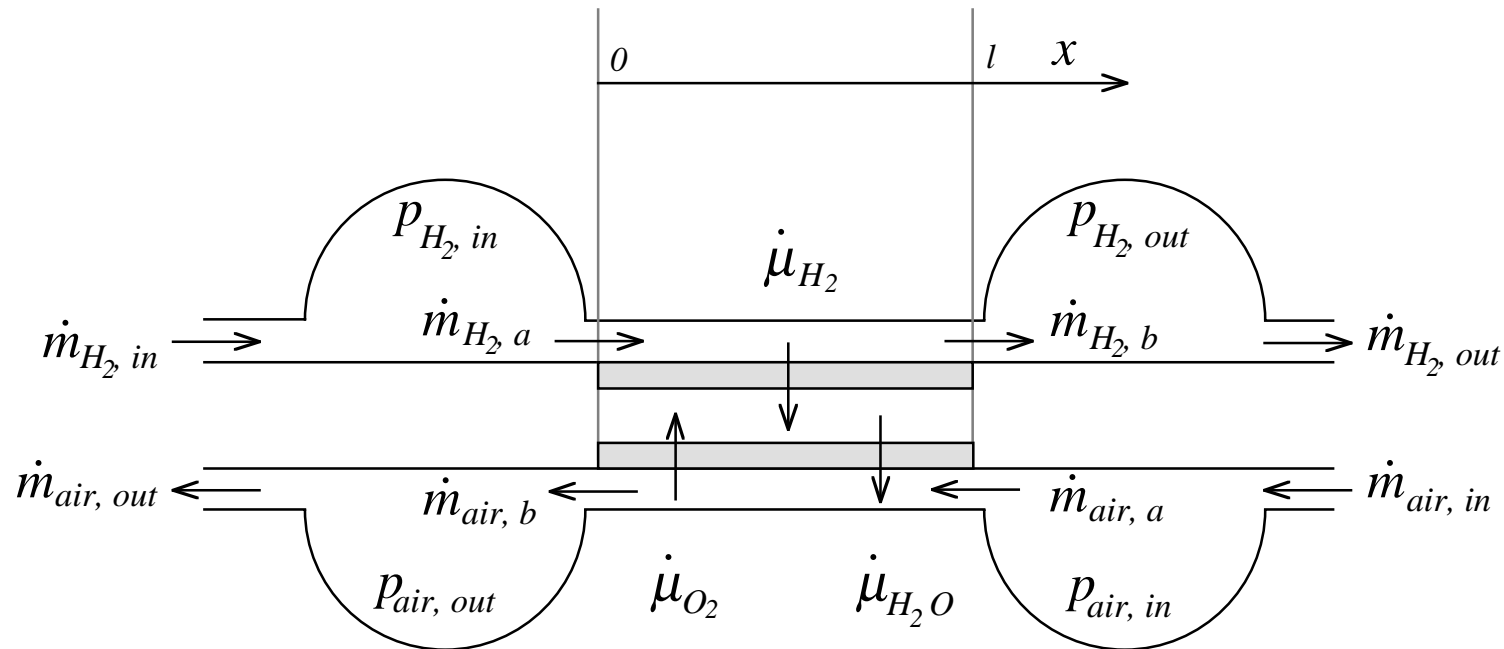
$O(10^0s)$ flow control/supercharging devices

$O(10^2s)$ cell and stack temperatures

Consequence:



- Manifolds described by classical "empty-and-filling" dynamics
- Between the manifolds *laminar flow* takes place in the many narrow grooves
- Massflow not constant along the x-direction (diffusion into/from the membrane)



$$\dot{\mu}_{H_2} = M_{H_2} I(t) / (2 \cdot F) \text{ etc.}$$

- Laminar flow-law

$$\dot{m}_{H_2}(x, t) = -k_{H_2} \cdot \frac{\partial p_{H_2}(x, t)}{\partial x}$$

- Flow incompressible but not constant due to the electrochemistry; assume uniform reaction intensity $\dot{\mu}_{\dots}$ along x

$$\dot{m}_{H_2}(x, t) = \dot{m}_{H_2, a}(t) - \frac{x}{l} \cdot \dot{\mu}_{H_2}(t)$$

- Resulting pressure profile

$$p_{H_2}(x, t) = p_{H_2, in}(t) - \frac{x}{k_{H_2}} \cdot \dot{m}_{H_2, a}(t) + \frac{x^2}{2 k_{H_2} l} \dot{\mu}_{H_2}$$

- Resulting mass-flows

$$\dot{m}_{H_2, a}(t) = \frac{k_{H_2}}{l} (p_{H_2, in}(t) - p_{H_2, out}(t)) + \frac{1}{2} \cdot \dot{\mu}_{H_2}(t)$$

$$\dot{m}_{H_2, b}(t) = \frac{k_{H_2}}{l} (p_{H_2, in}(t) - p_{H_2, out}(t)) - \frac{1}{2} \cdot \dot{\mu}_{H_2}(t)$$